WHAT IS CLAIMED IS:

1. A power roller supporting structure for a toroidal continuously variable transmission, the power roller supporting structure comprising:

a moveable disk rotatable about a first axis and axially moveable;

a stationary disk rotatable about the first axis and axially fixed, each of the moveable and stationary disks having a contact surface;

power rollers disposed between the moveable and stationary disks in contact with the contact surfaces thereof, each of the power rollers being rotatable about a second axis and pivotally moveable about a third axis extending perpendicular to the second axis upon rotation of the moveable and stationary disks, the power rollers having a friction contact position relative to the moveable and stationary disks in which a speed ratio between rotational speeds of the moveable and stationary disks is determined, the friction contact position including a low speed ratio position where a reduced speed ratio is obtained and a high speed ratio position where an increased speed ratio is obtained; and

an arrangement for allowing the power rollers to move closer to the first axis along the second axis when the power rollers are placed in the low speed ratio position than when the power rollers are placed in the high speed ratio position.

2. A power roller supporting structure for a toroidal continuously variable transmission, the power roller supporting structure comprising:

a moveable disk rotatable about a first axis and axially moveable;

a stationary disk rotatable about the first axis and axially fixed, each of the moveable and stationary disks having a contact surface;

power rollers disposed between the moveable and stationary disks in contact with the contact surfaces thereof, each of the power rollers being rotatable about a second axis and pivotally moveable about a third axis extending perpendicular to the second axis upon rotation of the moveable and stationary disks, the power rollers having a friction contact position relative to the moveable and stationary disks in which a speed ratio between rotational speeds of the moveable and stationary disks is determined, the friction contact position including a low speed ratio position where a reduced speed ratio is obtained and a high speed ratio position where an increased speed ratio is obtained;

trunnions supporting the power rollers so as to allow the rotation of the power rollers about the second axis and the pivotal movement thereof about the third axis, each of the trunnions having end portions opposed in a direction of the third axis, wherein the arrangement comprises:

a first link including a first periphery defining a first trunnion connection hole engaged with one of the end portions of each of the trunnion; and

a second link including a second periphery defining a second trunnion connection hole engaged with the other of the end portions of each of the trunnions;

at least one of the first and second trunnion connection holes being arranged to allow the power rollers to move closer to the first axis along the second axis when the power rollers are placed in the low speed ratio position than when the power rollers are placed in the high speed ratio position.

3. The power roller supporting structure as claimed in claim 2, wherein at least one of the first and second trunnion connection holes has a center which is offset from a center of curvature of the contact surface of the stationary disk in such a direction as to be away from the first axis along the second axis when the power rollers are placed in the high speed ratio position, in case that the at least one of the first and second connection holes and the second axis are projected onto a plane which extends perpendicular to the third axis and contains the first axis and the center of curvature of the contact surface of the stationary disk.

4. The power roller supporting structure as claimed in claim 2, wherein at least one of the first and second trunnion connection holes has a center which is offset from a center of curvature of the contact surface of the stationary disk in such a direction as to come close to the first axis along the second axis when the power rollers are placed in the low speed ratio position, in case that the at least one of the first and second trunnion connection holes and the second axis are projected onto a plane which extends perpendicular to the third axis and contains the first axis and the center of curvature of the contact surface of the stationary disk.

5. The power roller supporting structure as claimed in claim 2, wherein at least one of the first and second trunnion connection holes has a generally circular shape, the at least one of the first and second peripheries including a radius increasing portion at which the trunnion connection hole is increased in radius in such a direction

as to be away from the first rotation axis along the second rotation axis placed in the high speed ratio position, in case that the at least one of the first and second peripheries of the first and second links and the second axis are projected onto a plane which extends perpendicular to the third axis and contains the first axis and a center of curvature of the contact surface of the stationary disk,

the radius increasing portion including a high-side bearing portion which is pressed against the end portion of the trunnion when the power rollers are placed in the high speed ratio position, and a low-side bearing portion which is pressed against the end portion of the trunnion when the power rollers are placed in the low speed ratio position,

the high-side bearing portion being located more distant from the center of curvature of the contact surface of the stationary disk than the low-side bearing portion.

6. The power roller supporting structure as claimed in claim 5, wherein the radius increasing portion includes two sectoral regions disposed adjacent to each other on both sides of the second axis within the plane when the power rollers are placed in the high speed ratio position, each of the sectoral regions having a central angle of substantially 90 degrees.

7. The power roller supporting structure as claimed in claim 2, wherein at least one of the first and second trunnion connection holes has a generally circular shape, the at least one of the first and second peripheries including a radius reducing portion at which the trunnion connection hole is reduced in radius in such a direction as to come close to the first rotation axis along the second rotation axis placed in the low speed ratio position, in

case that the at least one of the first and second peripheries of the first and second links and the second axis are projected onto a plane which extends perpendicular to the third axis and contains the first axis and a center of curvature of the contact surface of the stationary disk,

the radius reducing portion including a high-side bearing portion which is pressed against the end portion of the trunnion when the power rollers are placed in the high speed ratio position, and a low-side bearing portion which is pressed against the end portion of the trunnion when the power rollers are placed in the low speed ratio position,

the high-side bearing portion being located more distant from the center of curvature of the contact surface of the stationary disk than the low-side bearing portion.

8. The power roller supporting structure as claimed in claim 7, wherein the radius reducing portion includes two sectoral regions disposed adjacent to each other on both sides of the second axis in the plane when the power rollers are placed in the low speed ratio position, each of the sectoral regions having a central angle of substantially 90 degrees.

9. The power roller supporting structure as claimed in claim 2, wherein at least one of the first and second trunnion connection holes has a generally circular shape which has a diameter extending across a center of curvature of the contact surface of the stationary disk along the second axis when the power rollers are placed in each of the high and low speed ratio positions, a midpoint of the diameter being offset from the center of curvature of the contact surface of the stationary disk, in case that the trunnion connection hole having a generally circular shape

and the second axis are projected onto a plane which extends perpendicular to the third axis and contains the first axis and the center of curvature of the contact surface of the stationary disk.

10. The power roller supporting structure as claimed in claim 9, wherein the midpoint of the diameter is offset from the center of curvature of the contact surface of the stationary disk in such a direction as to be away from the first axis along the second axis when the power rollers are placed in the high speed ratio position.

11. The power roller supporting structure as claimed in claim 9, wherein the midpoint of the diameter is offset from the center of curvature of the contact surface of the stationary disk in such a direction as to come close to the first axis along the second axis when the power rollers are placed in the low speed ratio position.

12. The power roller supporting structure as claimed in claim 10, wherein the at least one of the first and second trunnion connection holes has a circular shape which is centered at the midpoint of the diameter.

13. The power roller supporting structure as claimed in claim 11, wherein the at least one of the first and second trunnion connection holes has a circular shape which is centered at the midpoint of the diameter.

14. The power roller supporting structure as claimed in claim 9, wherein the at least one of the first and second trunnion connection holes has a modified circular shape including a radially enlarged portion, the at least one of

the first and second peripheries of the first and second links including a radius increasing portion at which the trunnion connection hole is increased in radius in such a direction as to be away from the first rotation axis along the second rotation axis placed in the high speed ratio position upon viewing the plane, the radius increasing portion including a high-side bearing portion which is pressed against the end portion of the trunnion when the power rollers are placed in the high speed ratio position, and a low-side bearing portion pressed against the end portion of the trunnion when the power rollers are placed in the low speed ratio position, the high-side bearing portion being located more distant from the center of curvature of the contact surface of the stationary disk than the low-side bearing portion.

15. The power roller supporting structure as claimed in claim 9, wherein the at least one of the first and second trunnion connection holes has a modified circular shape including a radially reduced portion, the at least one of the first and second peripheries of the first and second links including a radius reducing portion at which the trunnion connection hole is reduced in radius in such a direction as to come close to the first rotation axis along the second rotation axis placed in the low speed ratio position upon viewing the plane, the radius reducing portion including a high-side bearing portion which is pressed against the end portion of the trunnion when the power rollers are placed in the high speed ratio position, and a low-side bearing portion which is pressed against the end portion of the trunnion when the power rollers are placed in the low speed ratio position, the high-side bearing portion being located more distant from the center of curvature of

the contact surface of the stationary disk than the low-side bearing portion.

16. The power roller supporting structure as claimed in claim 14, wherein the radius increasing portion includes two sectoral regions disposed adjacent to each other on both sides of the second axis within the plane when the power rollers are placed in the high speed ratio position, each of the sectoral regions having a central angle of substantially 90 degrees.

17. The power roller supporting structure as claimed in claim 15, wherein the radius reducing portion includes two sectoral regions disposed adjacent to each other on both sides of the second axis in the plane when the power rollers are placed in the low speed ratio position, each of the sectoral regions having a central angle of substantially 90 degrees.